



CFturbo

All-New Hydraulic Design of an Electrical Automotive Coolant Pump

by Sascha Henoch and Ralph Peter MUELLER

- TENGAM Engineering Inc., is an automotive supplier that manufactures injection-molded magnets
- The company has decided to enter a new market segment: **electric automotive coolant pump**
- In cooperation with CFturbo, Inc. an all-new product has been created, a pump with a brushless electric motor
- CFturbo was responsible for the hydraulic design, CFD simulations, and optimization.
- Design targets
 - Compact dimensions
 - High efficiency
 - Low noise



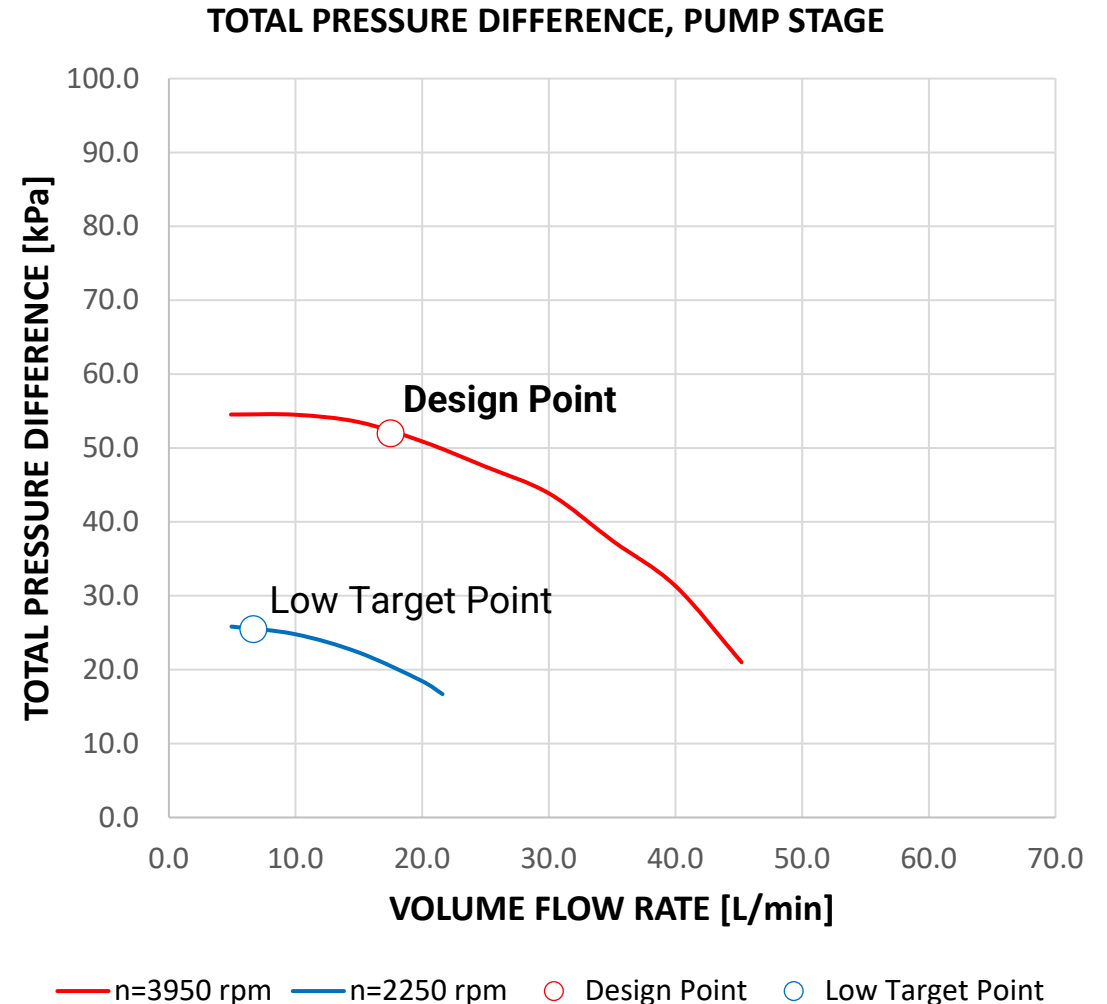
Fluid Glycol water mixture 50% - 50% at 20°C

Design point

Volume Flow Rate	17.5	L/min
Pressure Difference	52.0	kPa
Rotational Speed	3950	rev/min
Specific speed (EU)	20	

Low Target Point

Volume Flow Rate	6.67	L/min
Pressure Difference	25.5	kPa
Rotational Speed	2250	rev/min
Specific speed (EU)	12	



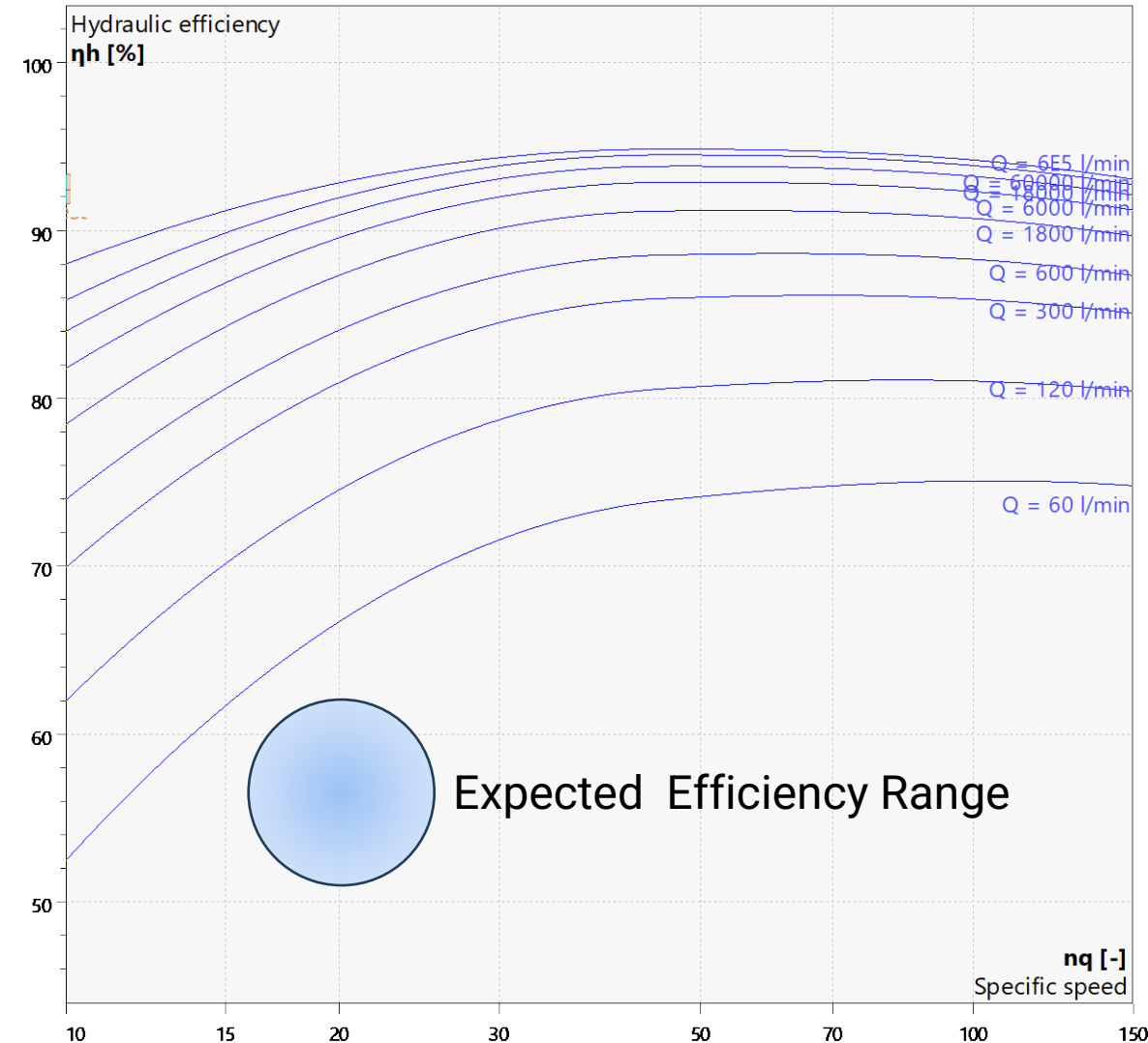
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- CFD plays a crucial role in modern product design in the automotive industry
- Frontloading simulation: implement CFD simulations from the very beginning in the development process
- Meet design targets
- Get optimal solutions
- Reduce product development time and cost

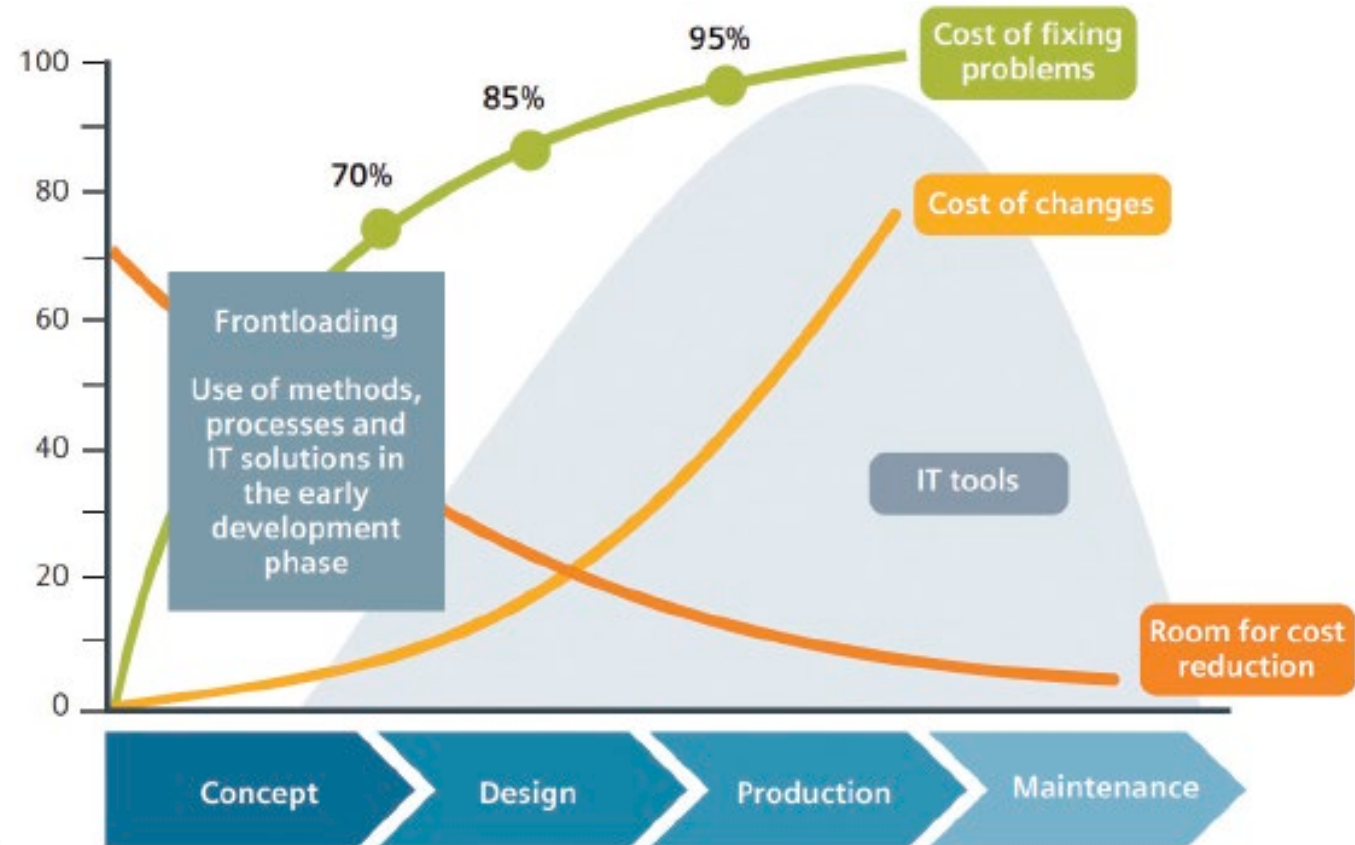
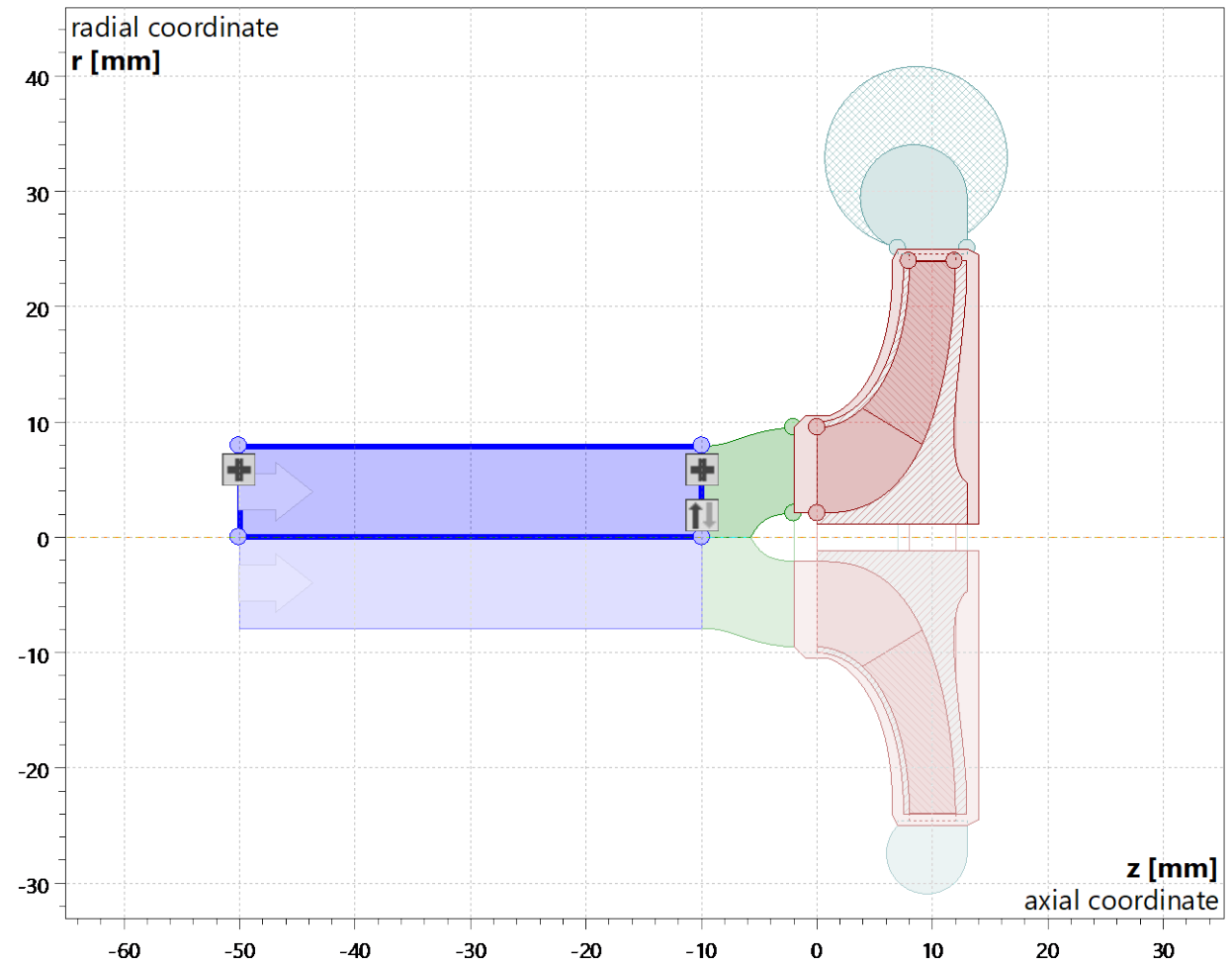
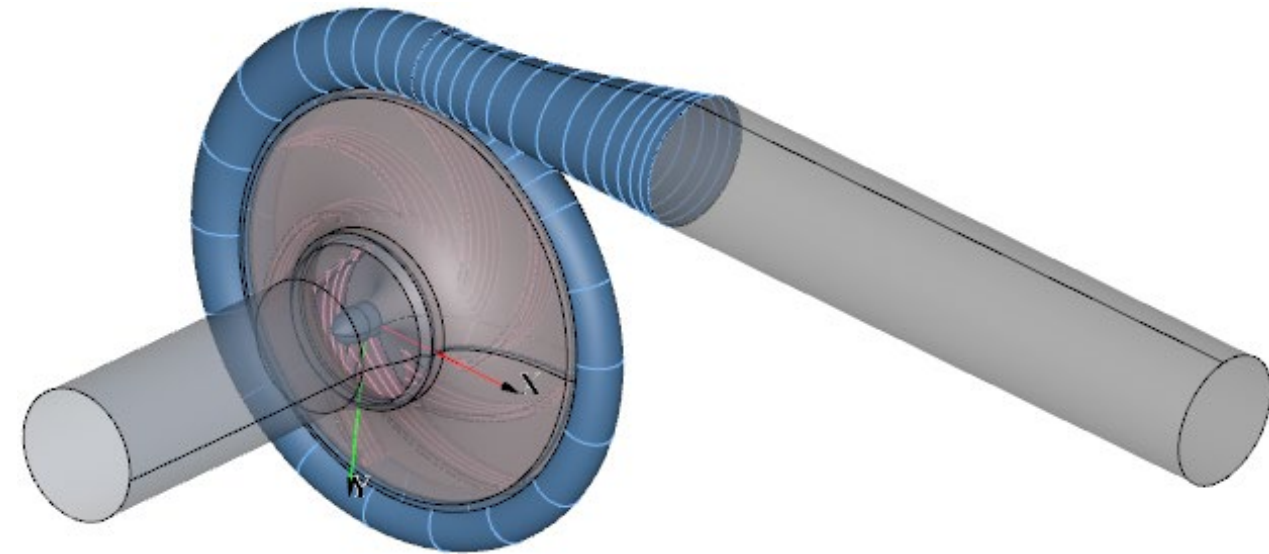


Image source: Prof. Dr. Martin Eigner, VPE, TU Kaiserslautern

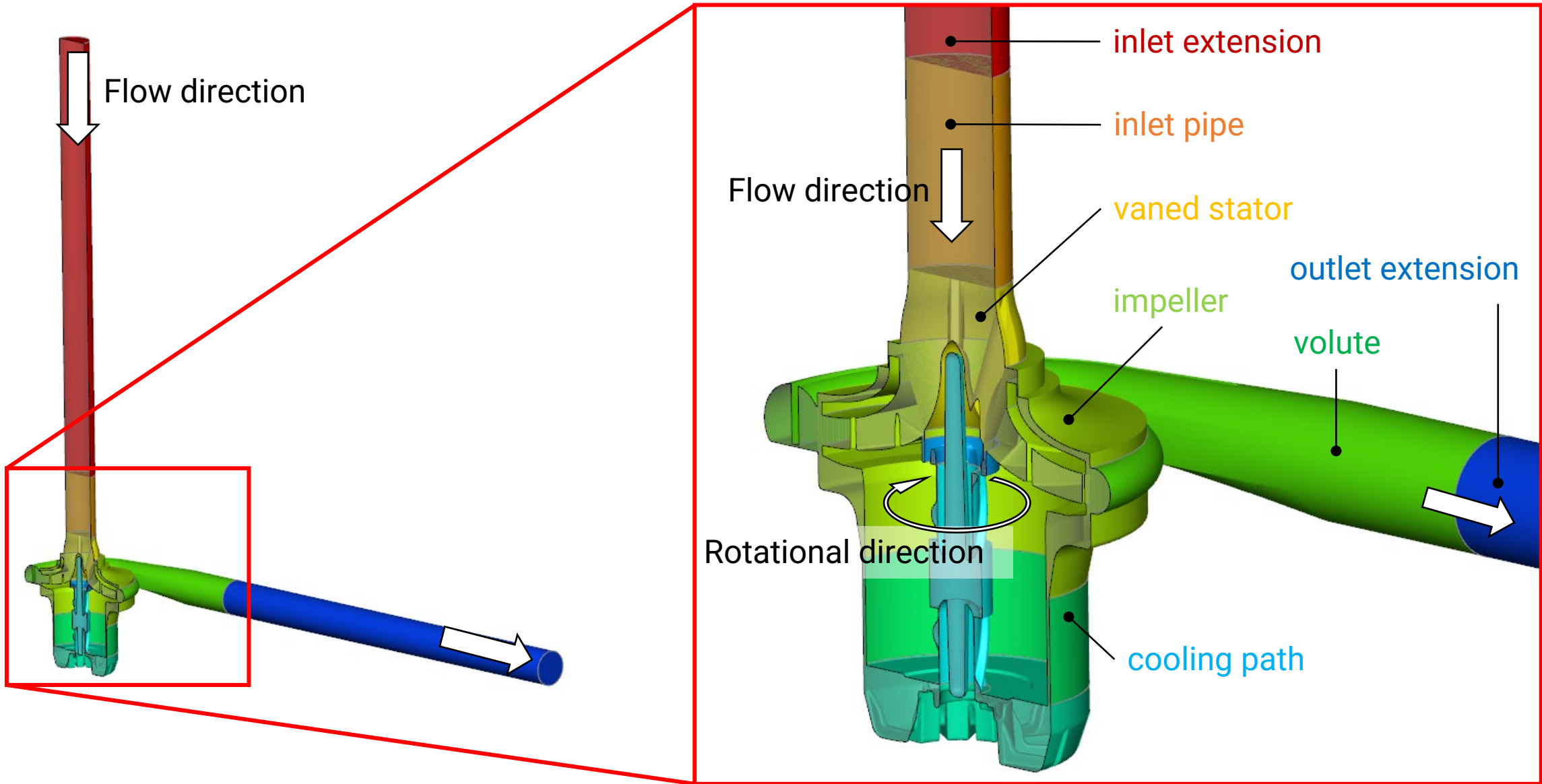
Initial design phase

Conceptual Pump Design

- Start with a reduced pump model
- Gradually increase complexity

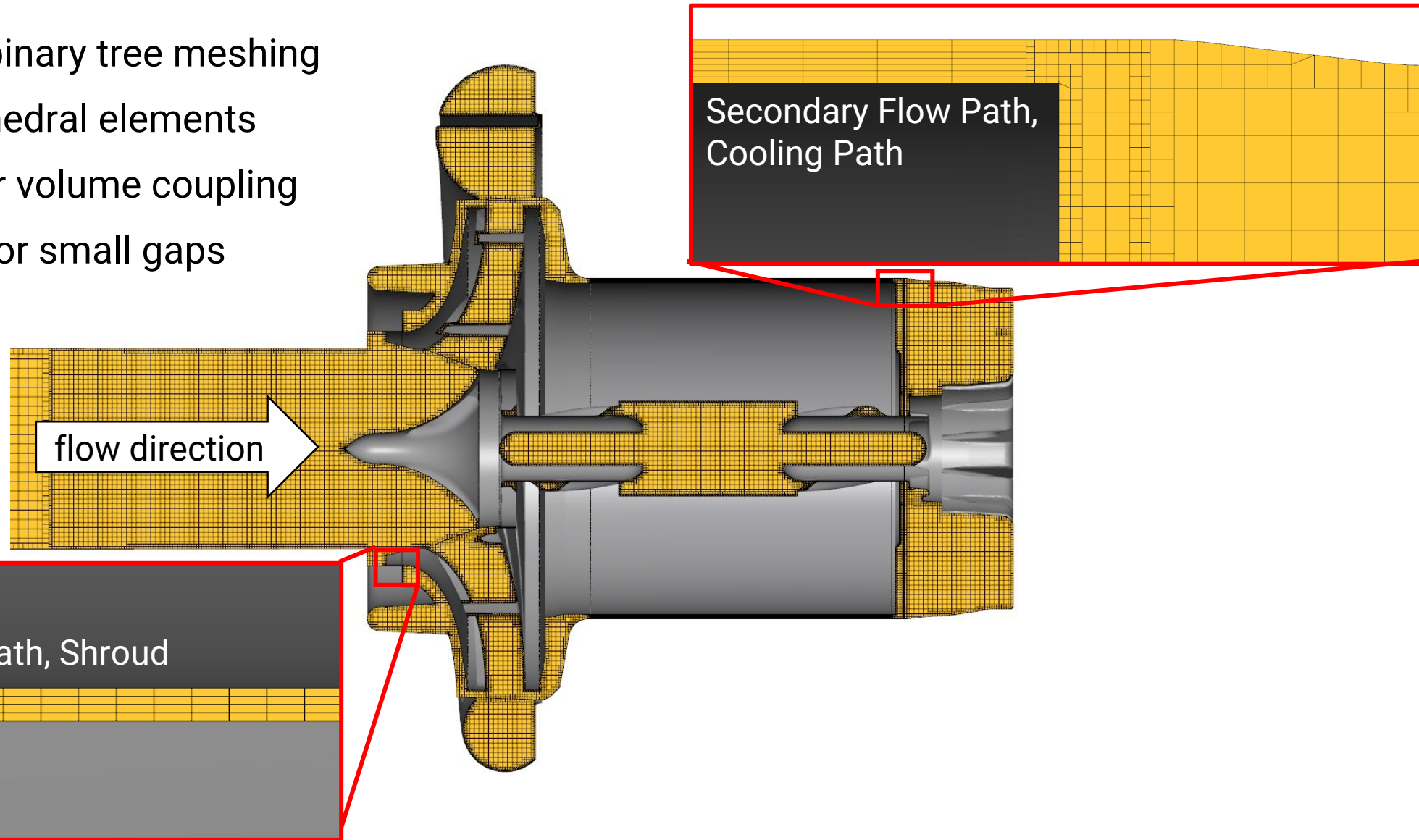


Model Setup

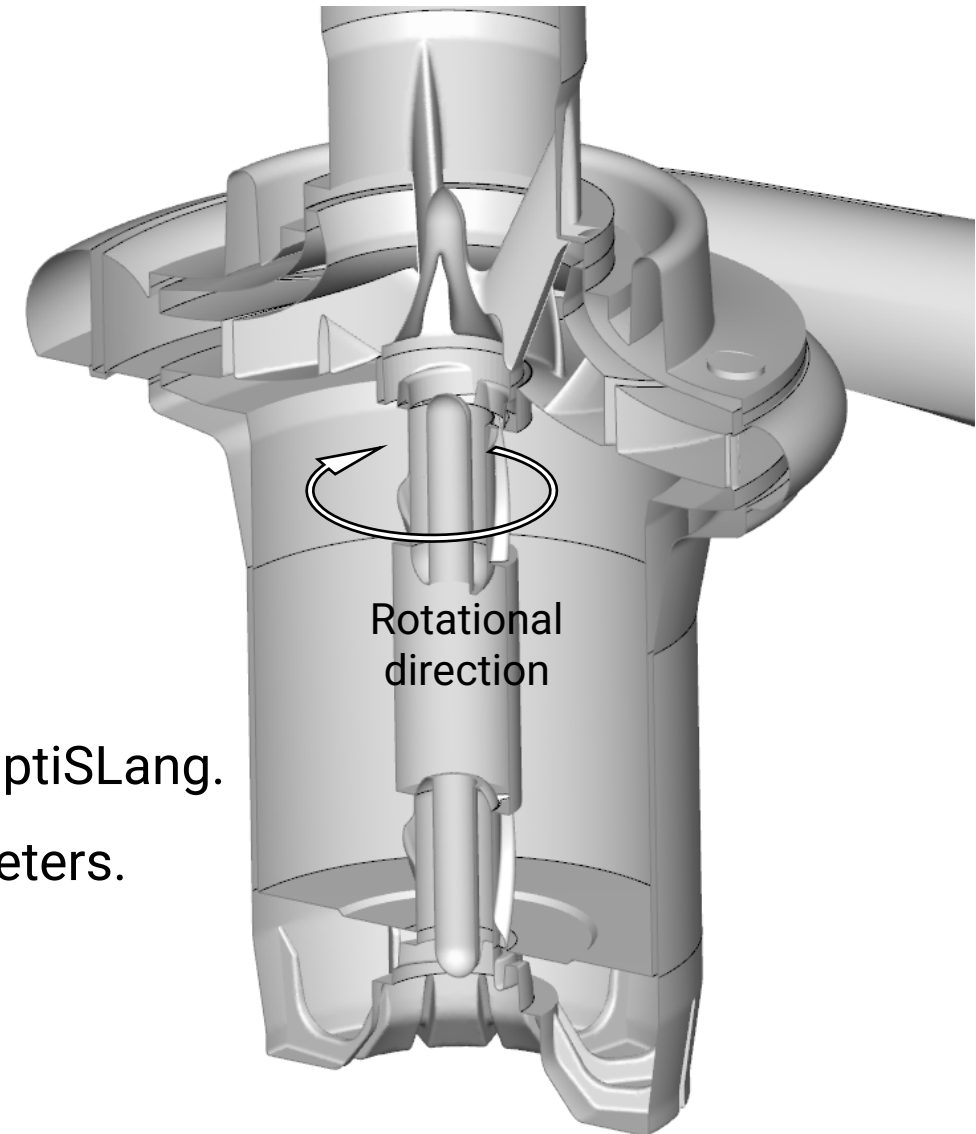


Computational Grid

- Cartesian mesh, binary tree meshing
- Dominantly hexahedral elements
- Grid interfaces for volume coupling
- Structured grids for small gaps
- 7.5 million nodes

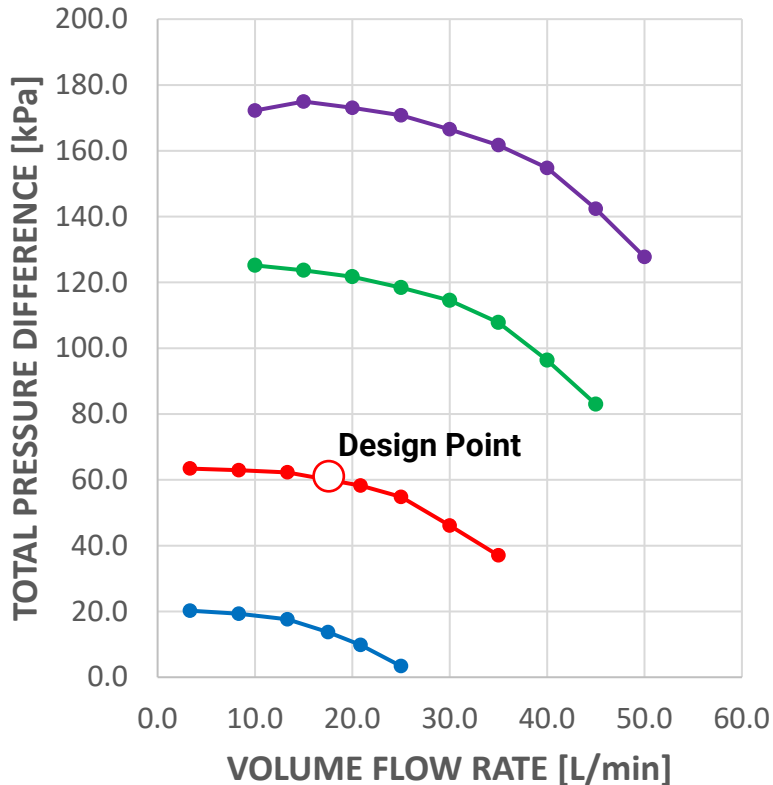


- Isothermal
- Steady-state (MFR) and transient simulations
- High-order discretization scheme (space, time)
- RNG-based k - ϵ turbulence model
- Unified wall function
- Hydraulic smooth surfaces
- **Design optimization:** CFturbo + Simerics-MP + Dakota/optiSLang.
- Statistical methods to identify the most sensitive parameters.
- Surrogate-assisted optimization.

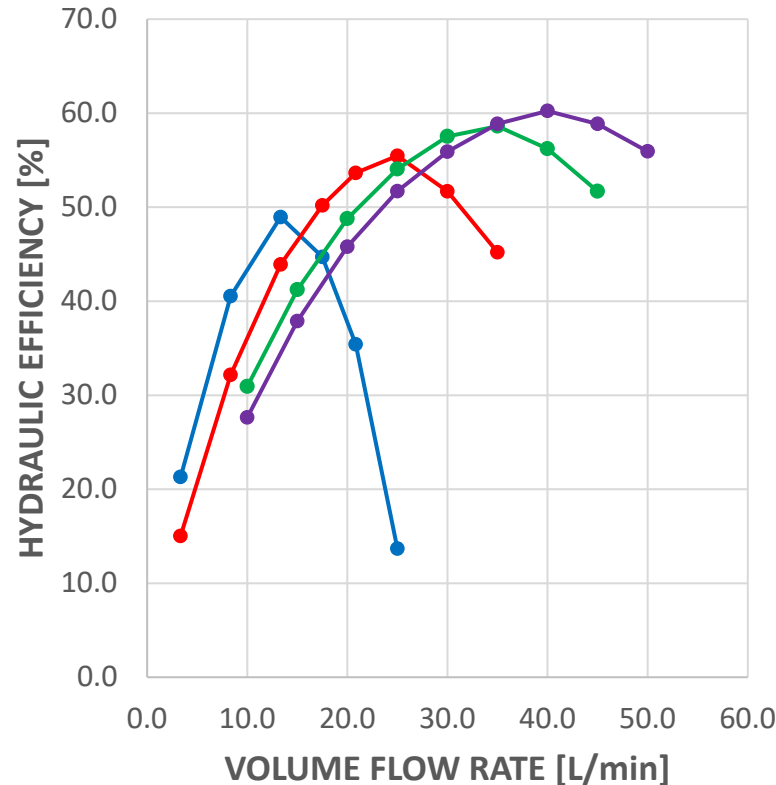


Transient flow, 4 impeller revolutions, 5 hours (AMD Ryzen Threadripper Pro 3945WX, 8 threads)

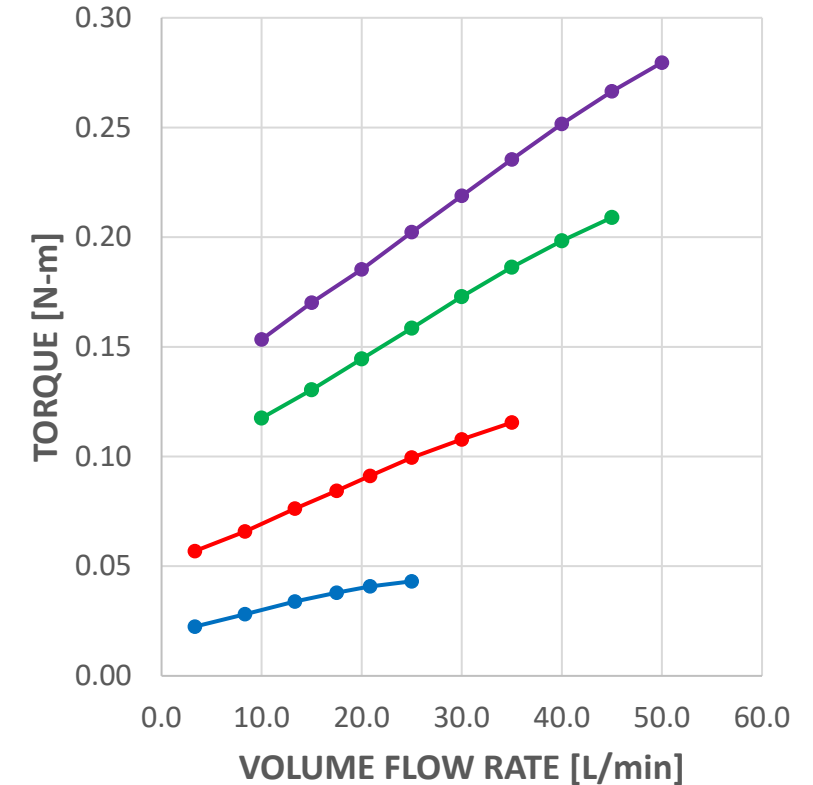
TOTAL PRESSURE DIFFERENCE, PUMP



HYDRAULIC EFFICIENCY, PUMP



TORQUE, IMPELLER

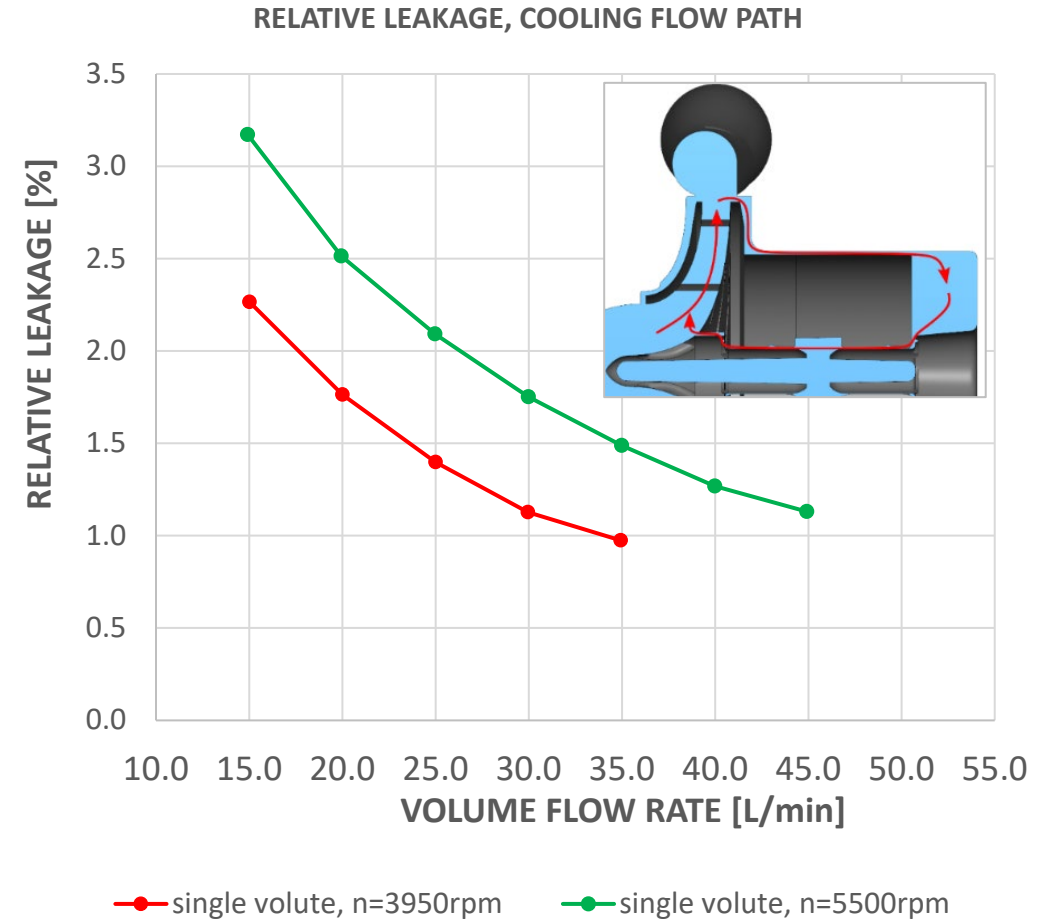
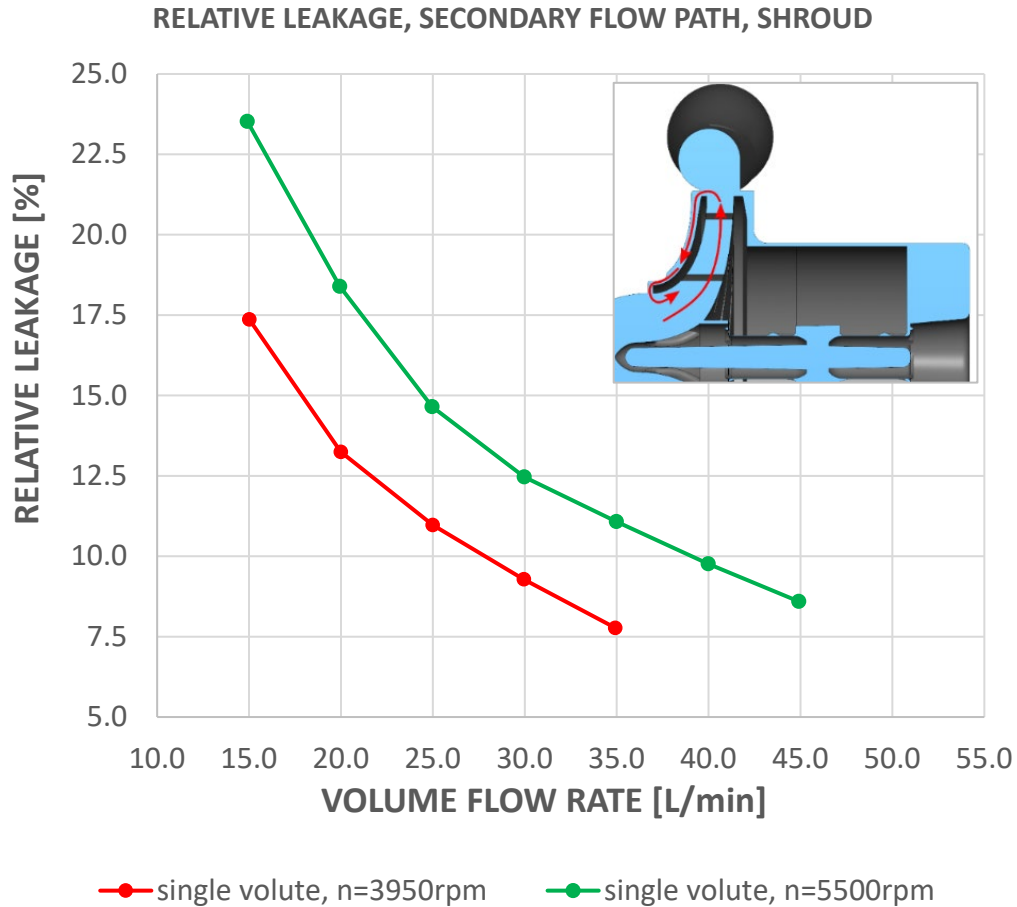


● n=2250rpm ● n=3950rpm
● n=5500rpm ● n=6500rpm

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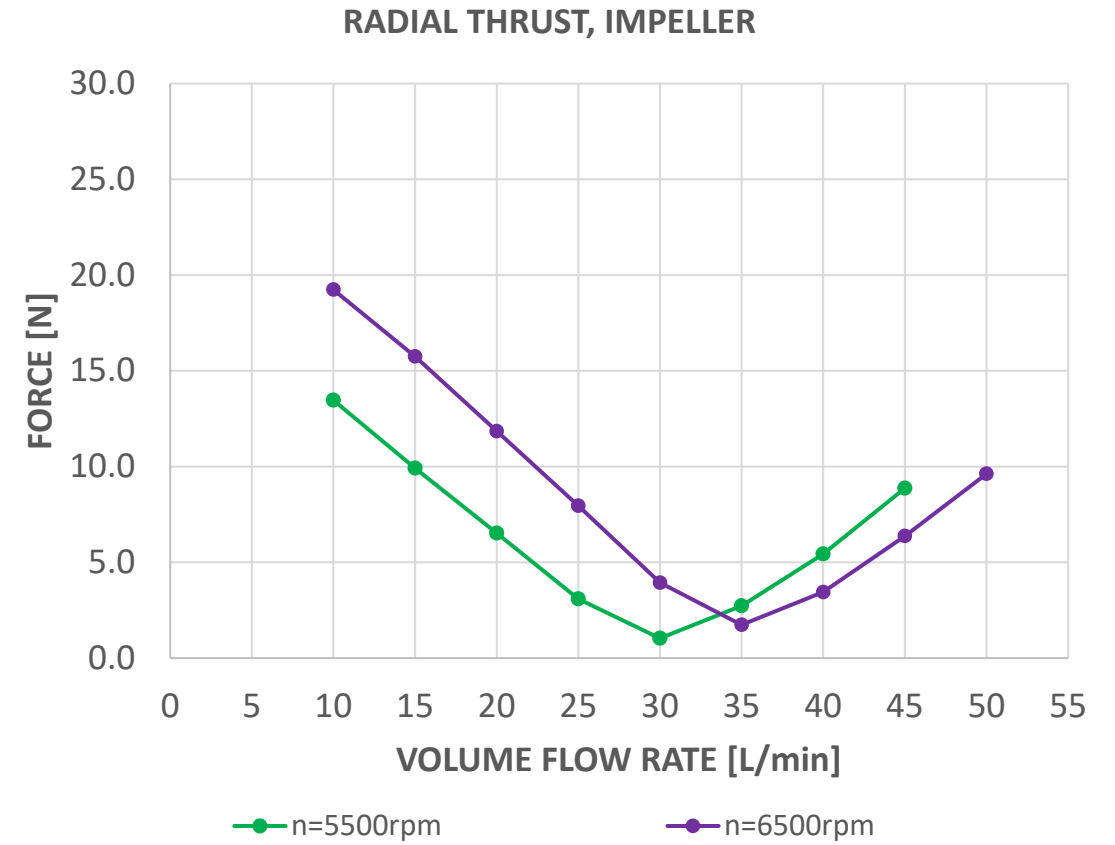
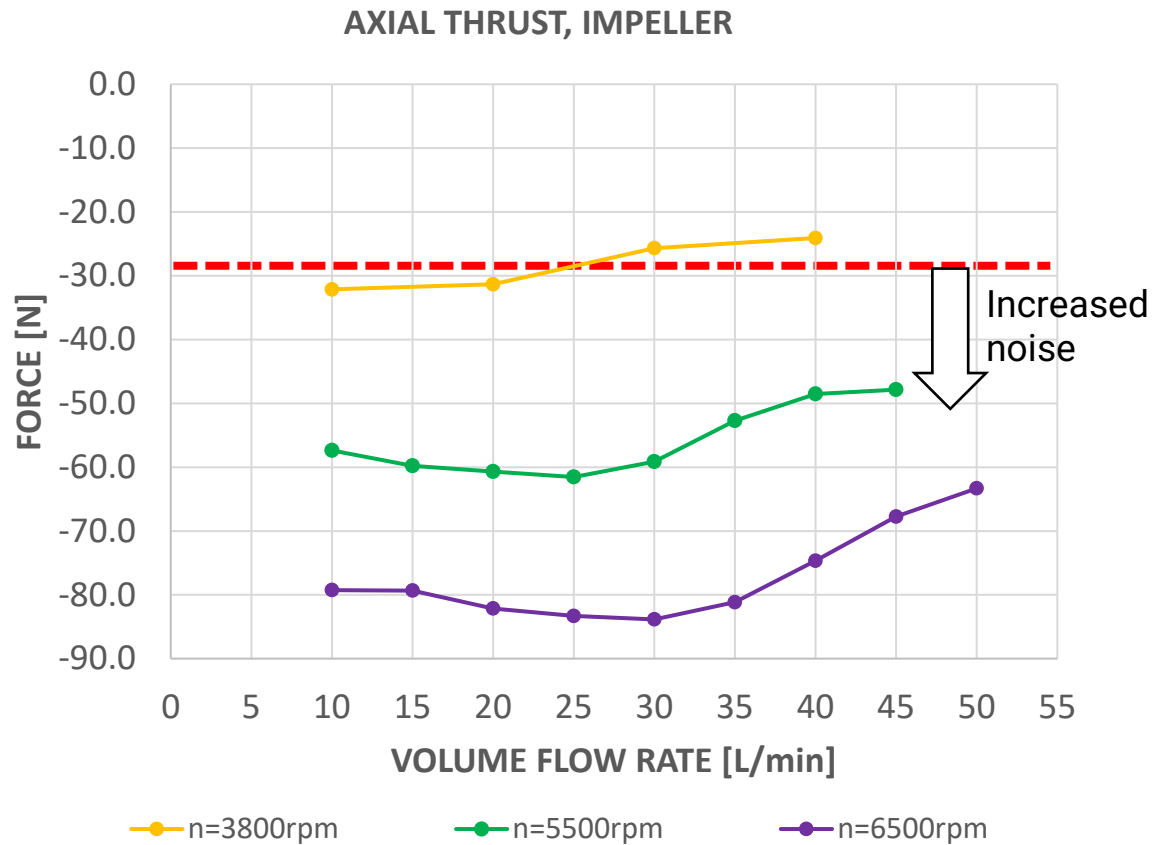
● n=2250rpm ● n=3950rpm
● n=5500rpm ● n=6500rpm

Leakage flow needs to be considered, even in the early development stage.

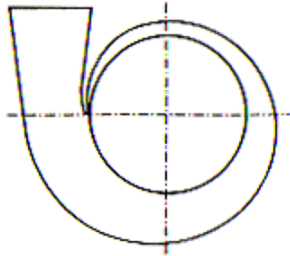


Prototype testing the initial design model → some operating points were noise-wise conspicuously

Noticeable wear at the axial thrust bearing was visible after prototype testing

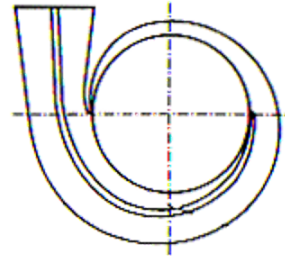


- **Double volute design** to limit radial force level in off-design points



Single volute

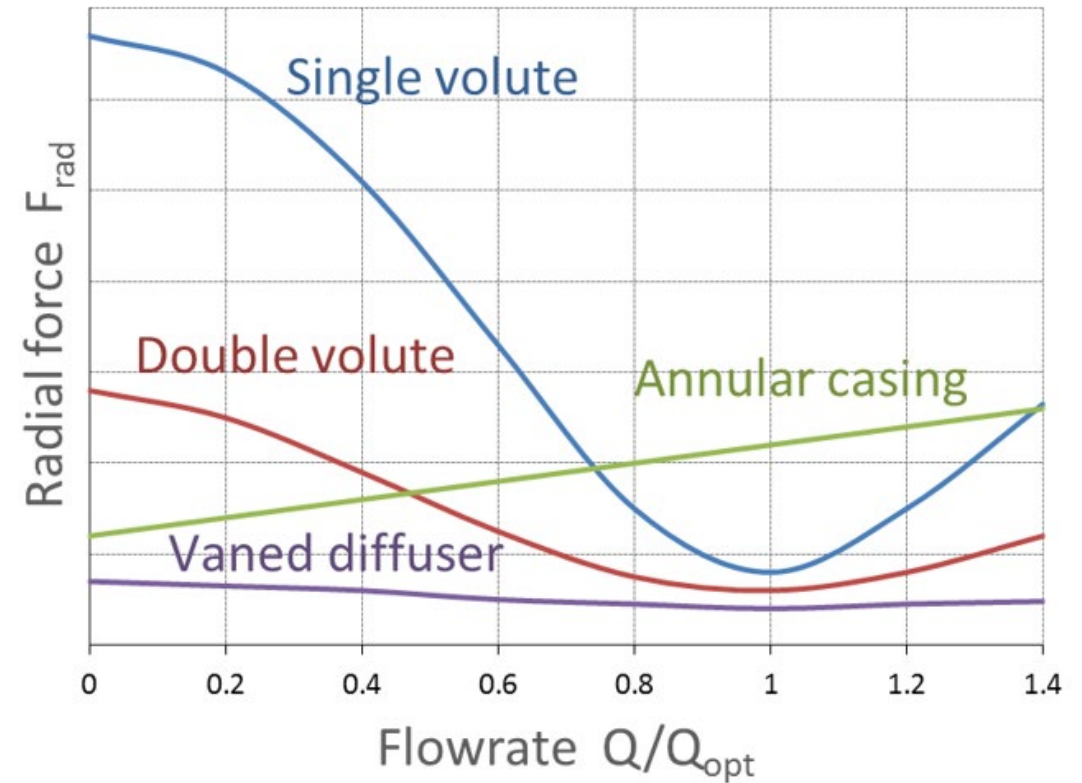
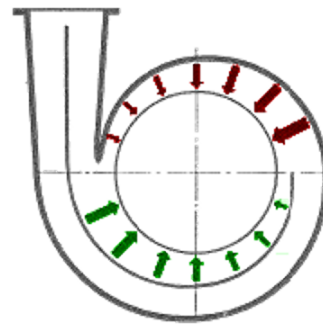
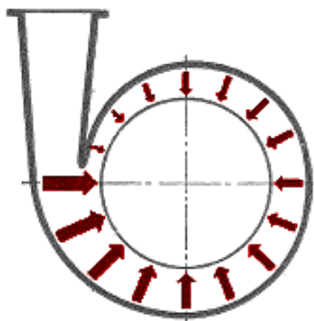
- Lowest production cost
- Preferred for low energy transmission



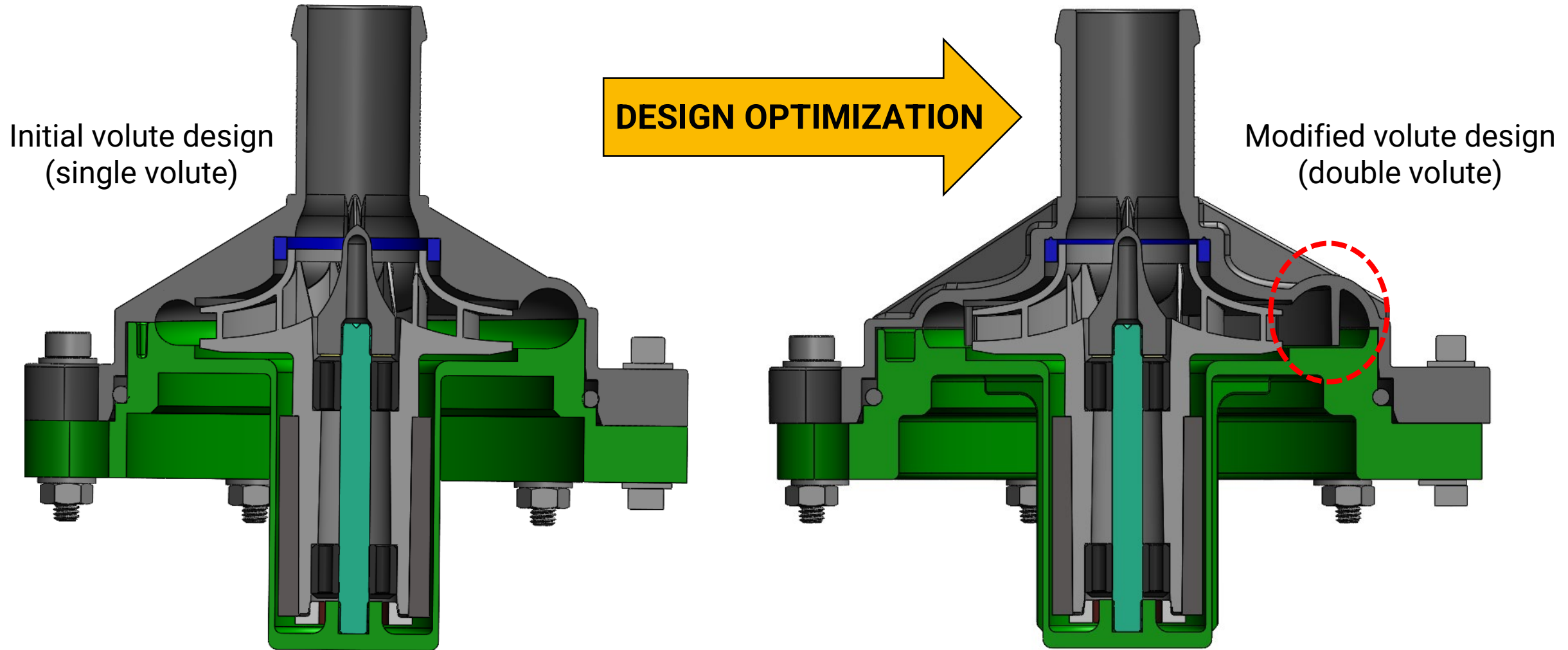
Double volute

- Reduction of radial forces
- Common pressure joint

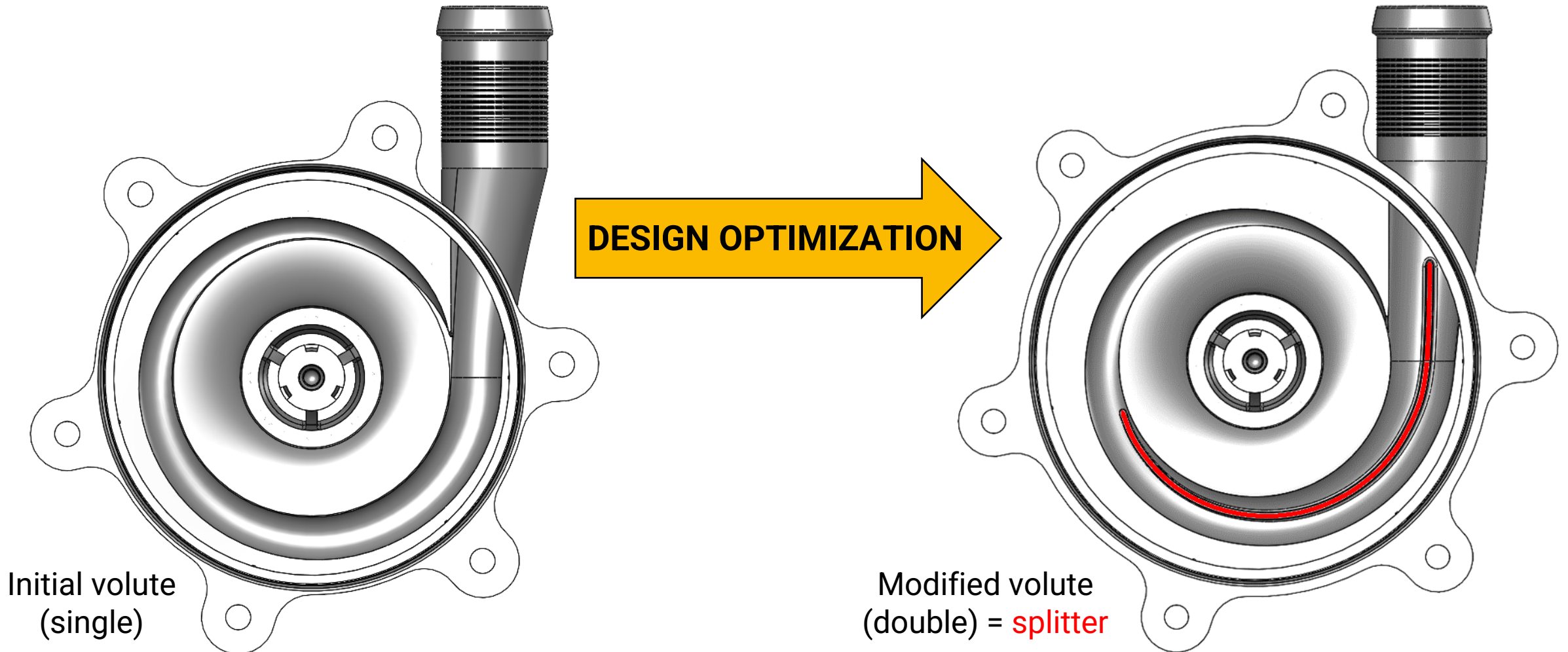
Radial force



Double volute design to limit radial force level in off design points



Double volute design to limit radial force level in off-design points

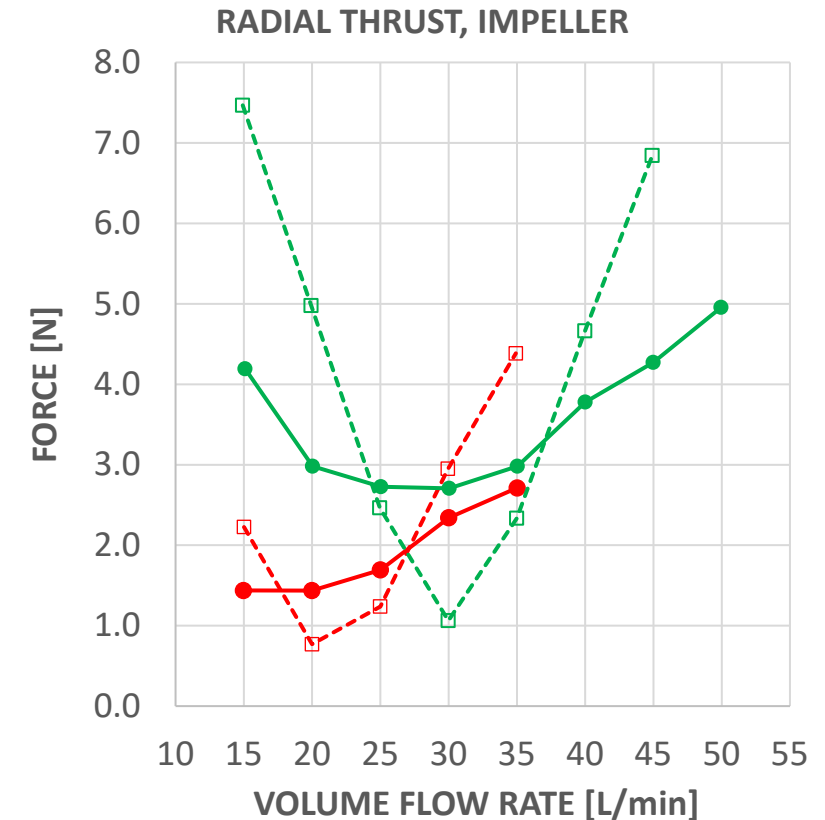
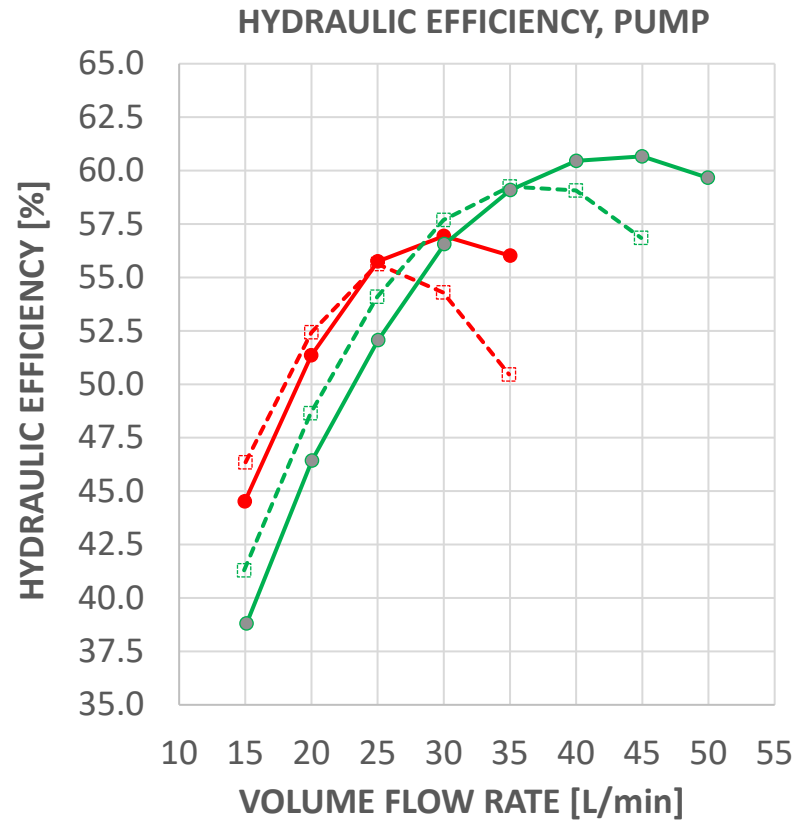
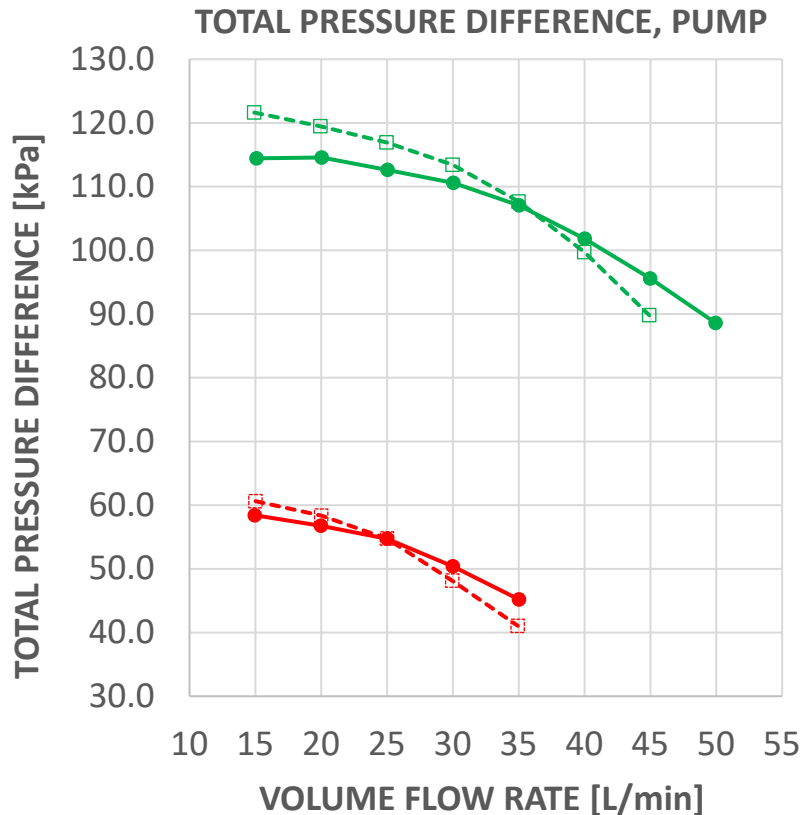


Initial volute
(single)

Modified volute
(double) = **splitter**

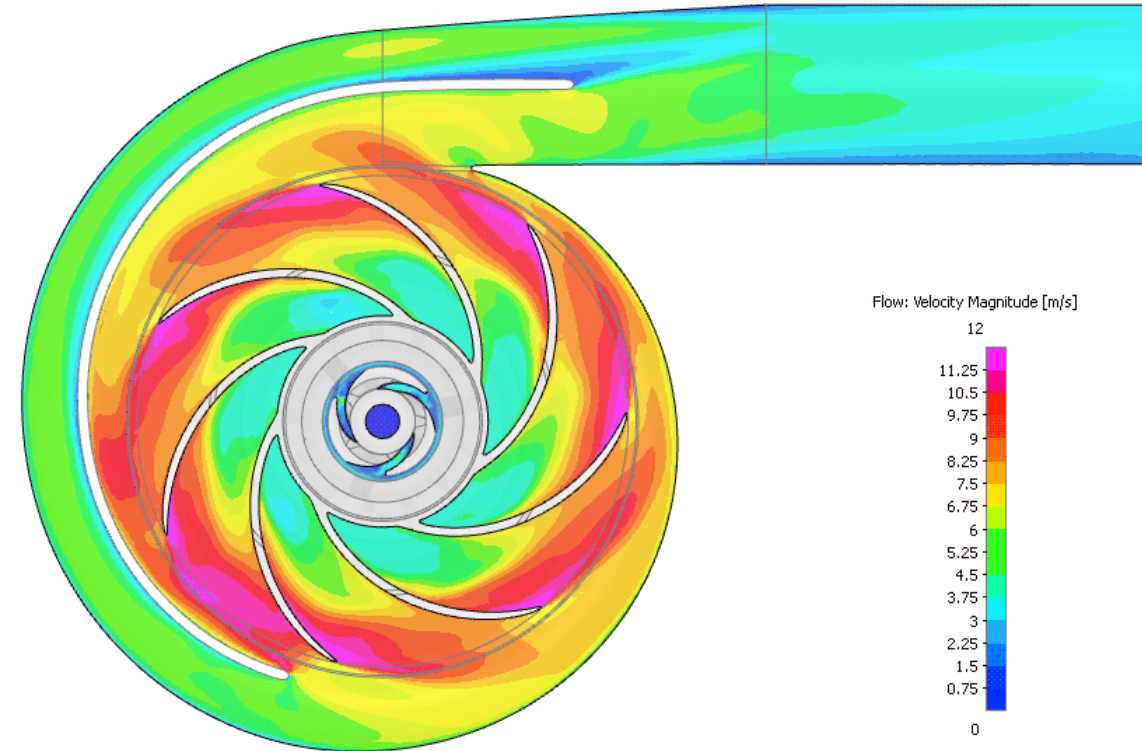
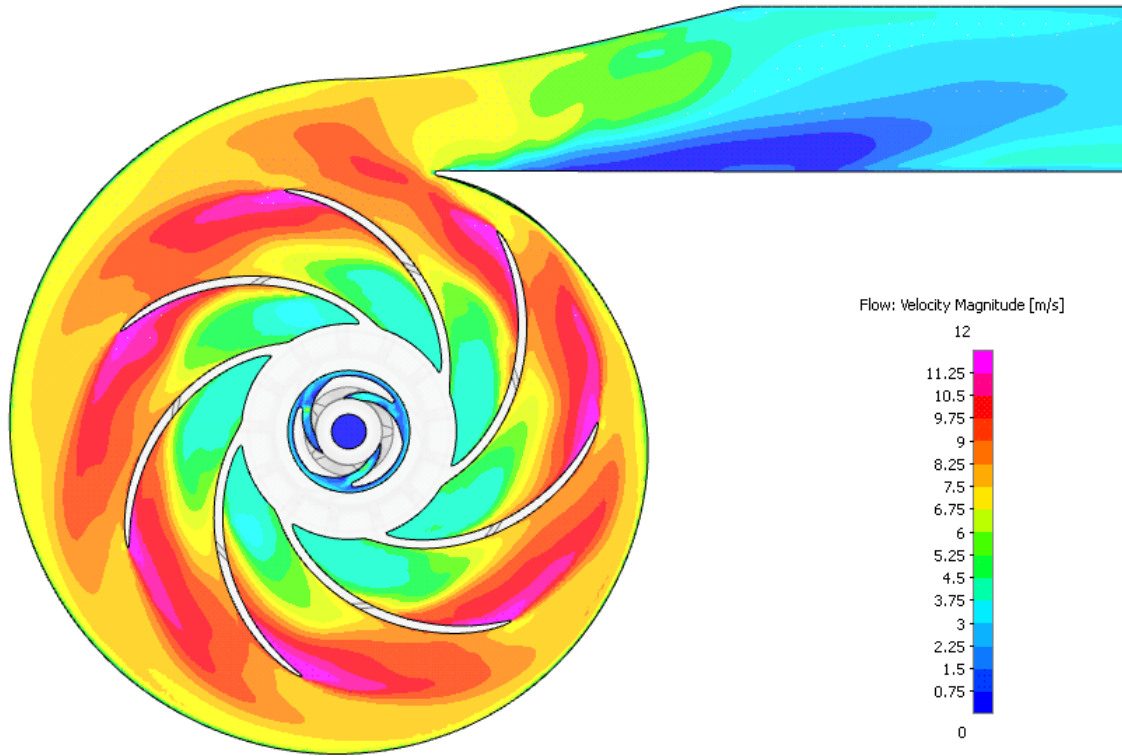
Test Results – Analysis and Solution

Volute with splitter → optimize design to maintain efficiency



Centrifugal Pump Simulation and Analysis

n=5500 rpm, Q=35 l/min



Design point adjustment

Design Point Adjustment

- A customer demand forced a **design point shift**, that needs substantially higher pump performance!

New Design point

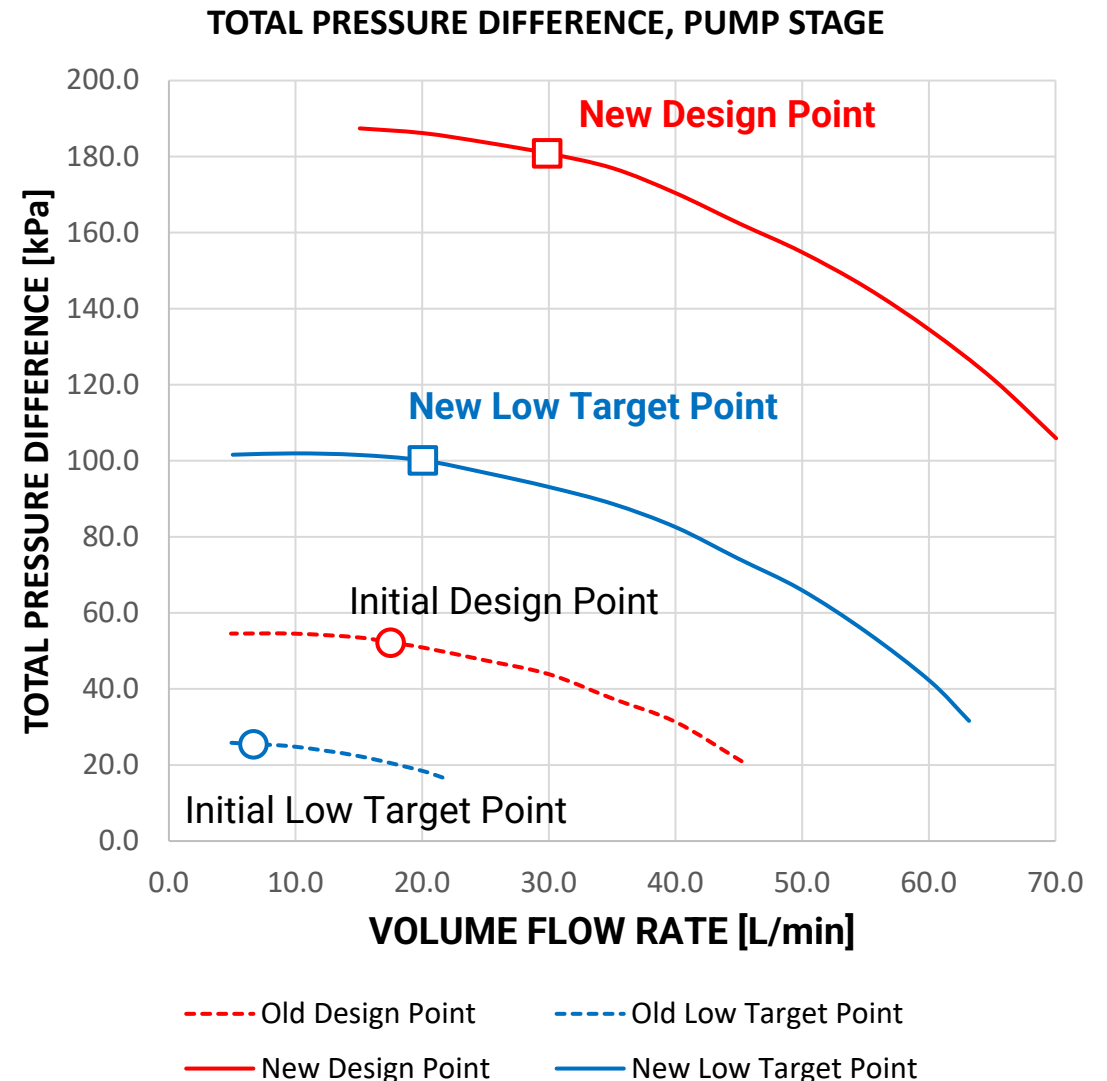
Volume Flow Rate	35.0	L/min
Pressure Difference	180.0	kPa

New Low Target Point

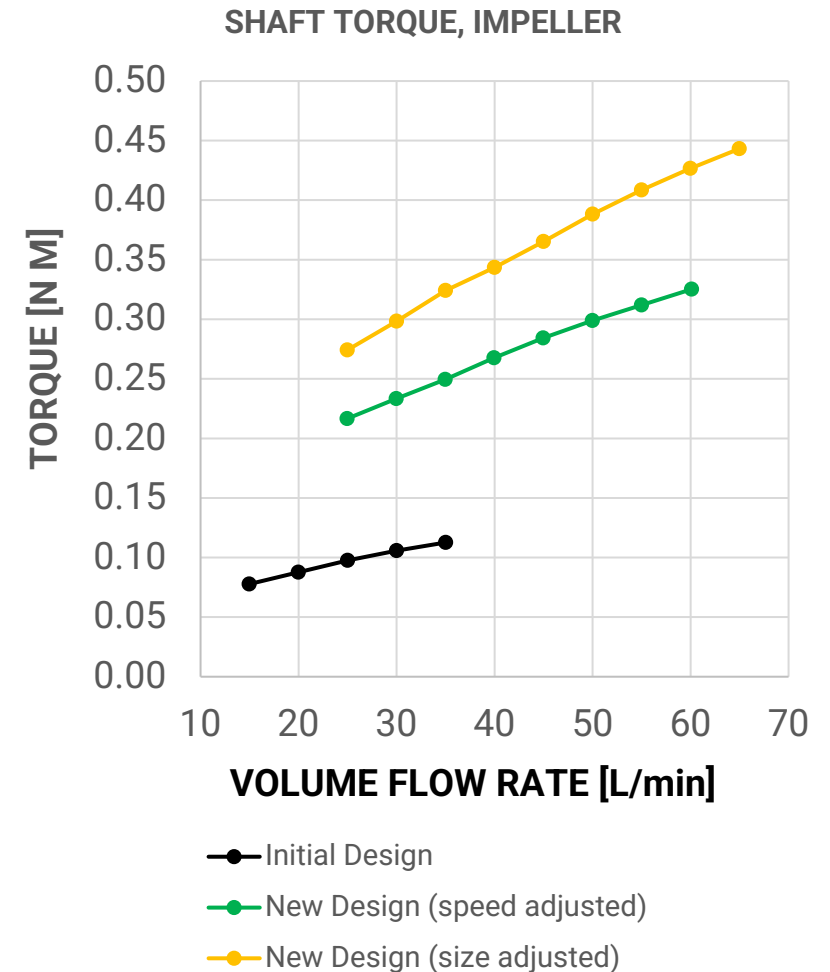
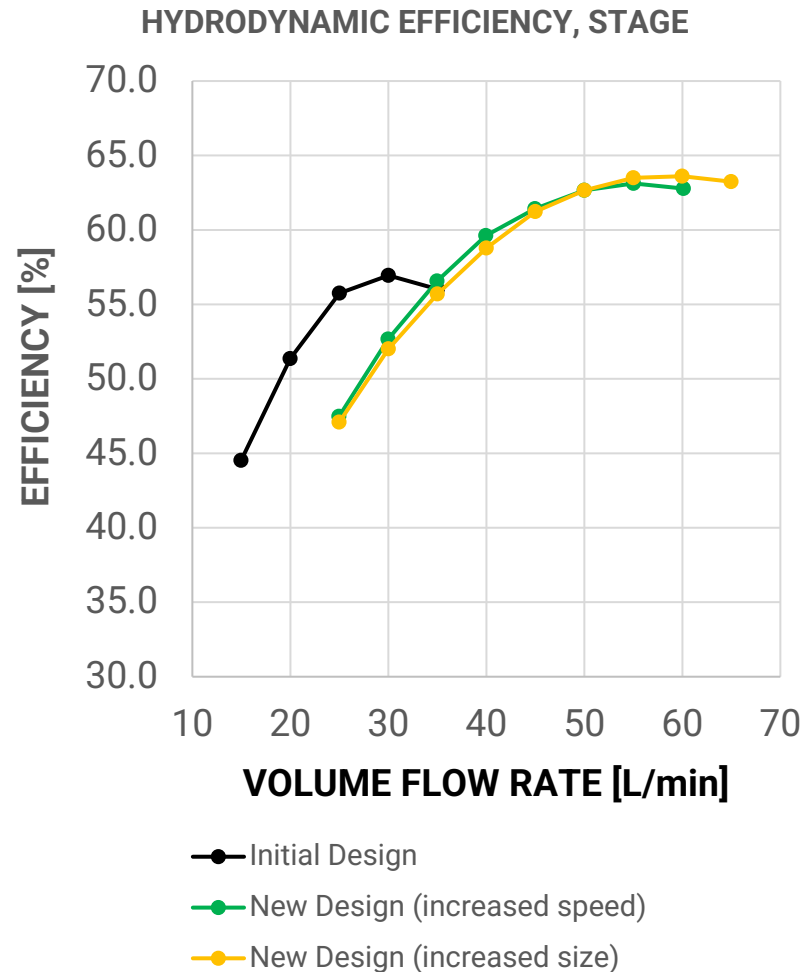
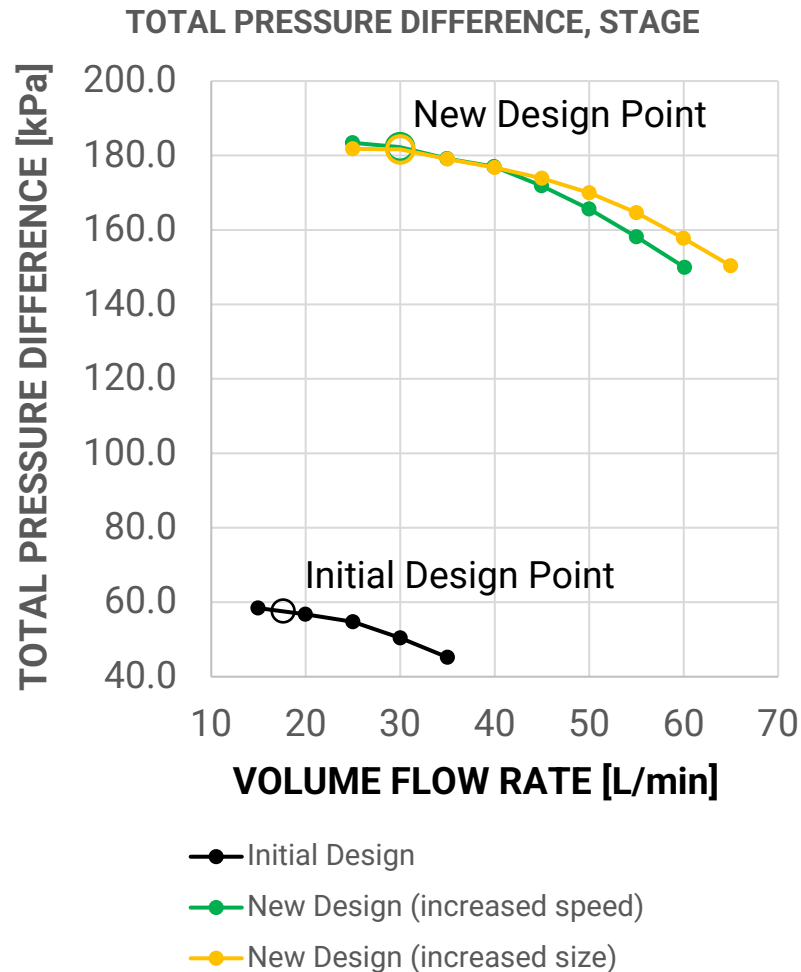
Volume Flow Rate	20.0	L/min
Pressure Difference	100.0	kPa



Re-design, design exploration, optimization.

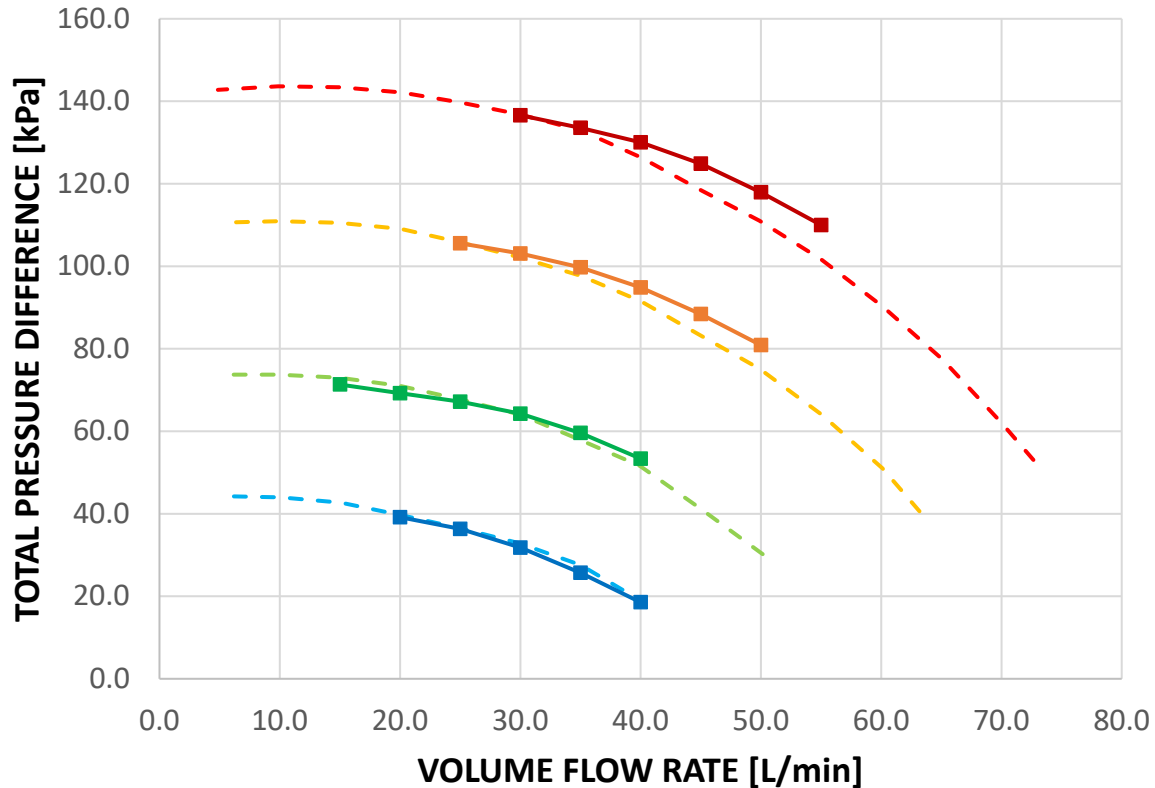


Using optimization, two new designs were identified for the adjusted design point.



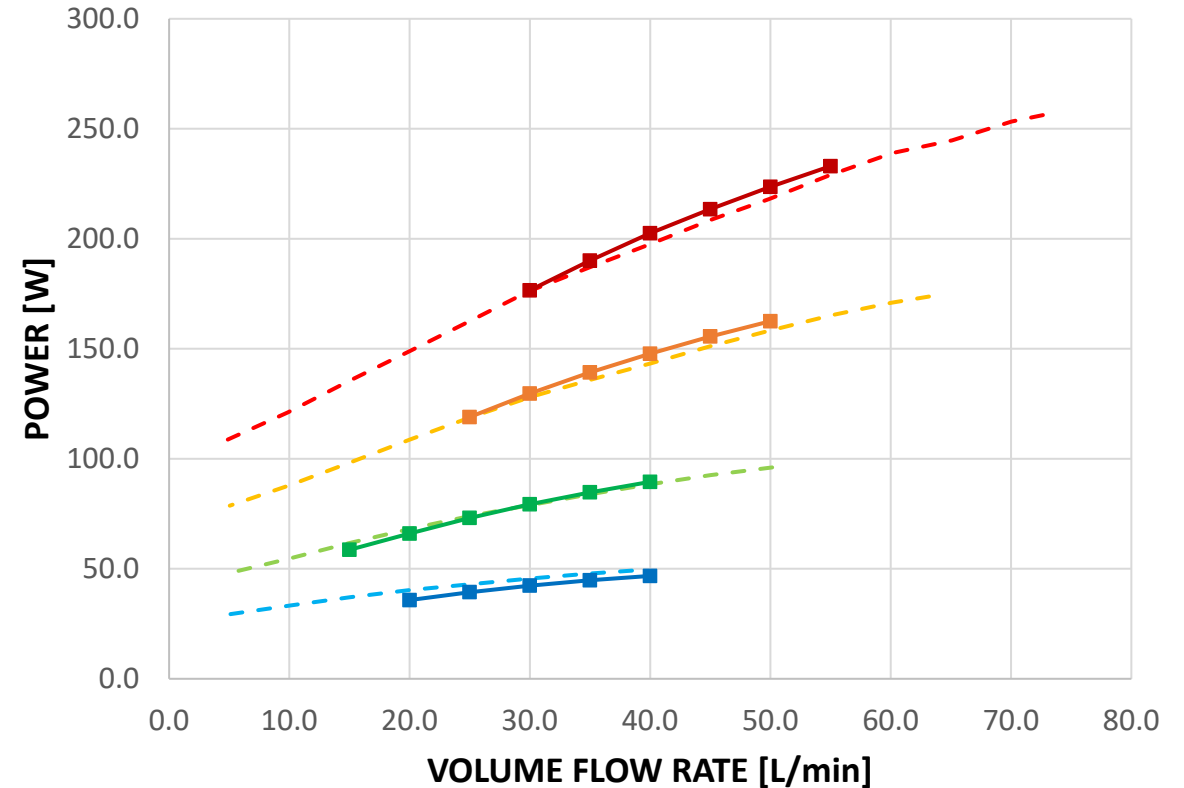
Manufacturable product

TOTAL PRESSURE DIFFERENCE, PUMP STAGE



- FlowStand,3400 RPM —■ CFD,transient,3400 RPM
- FlowStand,4400 RPM —■ CFD,transient,4400 RPM
- FlowStand,5400 RPM —■ CFD,transient,5400 RPM
- FlowStand,6150 RPM —■ CFD,transient,6150 RPM

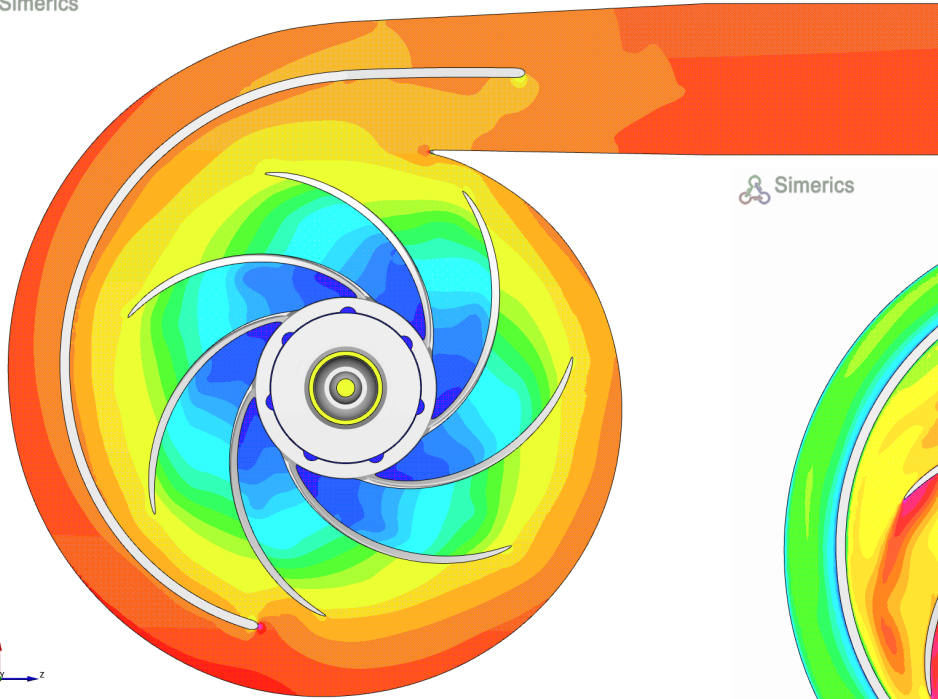
REQUIRED ELECTRICAL POWER, PUMP STAGE



- FlowStand,3400 RPM —■ CFD,transient,3400 RPM
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- FlowStand,5400 RPM —■ CFD,transient,5400 RPM
- FlowStand,6150 RPM —■ CFD,transient,6150 RPM

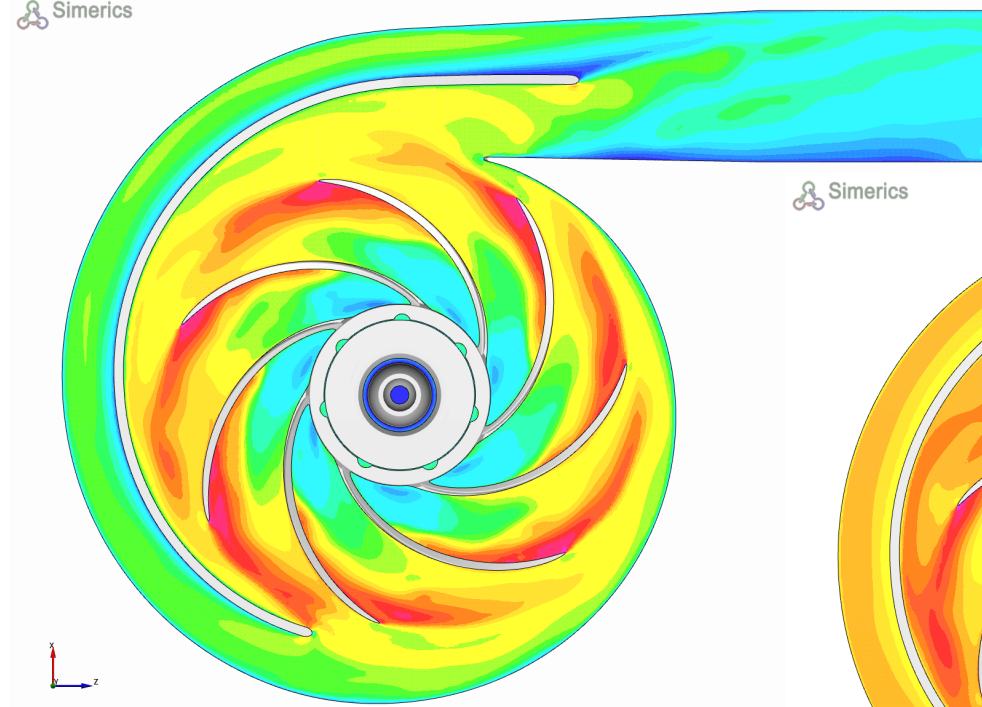
Market-ready Product – Test and CFD Results

Static Pressure



$n=6150$ rpm, $Q=45$ l/min

Velocity Magnitude



Total Pressure

